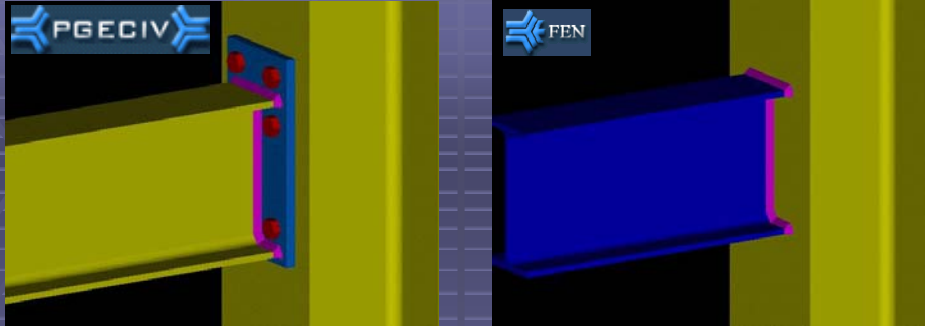




Ligações Soldadas

Terceira Parte



Programa de Pós-Graduação em Engenharia Civil

PGECIV - Mestrado Acadêmico

Faculdade de Engenharia – FEN/UERJ

Disciplina: Tópicos Especiais em Projeto (Ligações em Aço e Mistras)

Professor: Pedro Vellasco

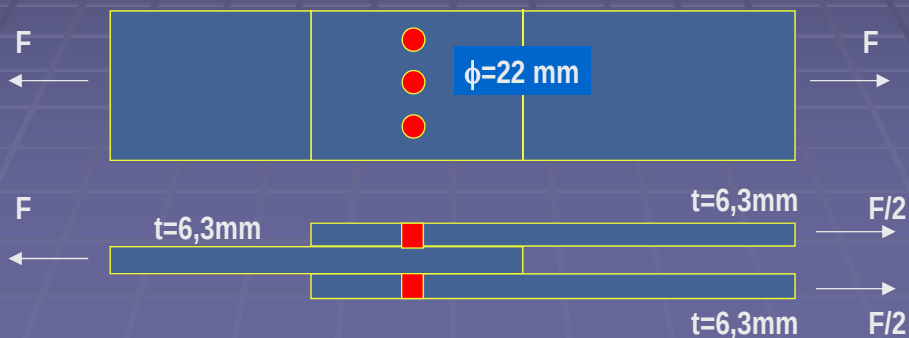
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14. Exemplos

- Determinar o valor mínimo para a força F sabendo-se que foi utilizada solda de bujão

- Dados: $F_y = 250 \text{ MPa}$ e $F_w = 415 \text{ MPa}$



3

14. Exemplos

$$V_r \text{ menor de } 0,67 \cdot \phi \cdot A_m \cdot F_y = 0,67 \cdot 0,9 \cdot 0,250 \cdot A_m = 0,15 A_m \leftarrow$$

$$0,67 \cdot \phi_w \cdot A_w \cdot F_w = 0,67 \cdot 0,67 \cdot 0,415 \cdot A_w = 0,19 A_m$$

$$A_m = A_w = \left(\frac{\pi d^2}{4} \right) \times 3 = \left(\frac{\pi (22)^2}{4} \right) \times 3 = 1140,4 \text{ mm}^2$$

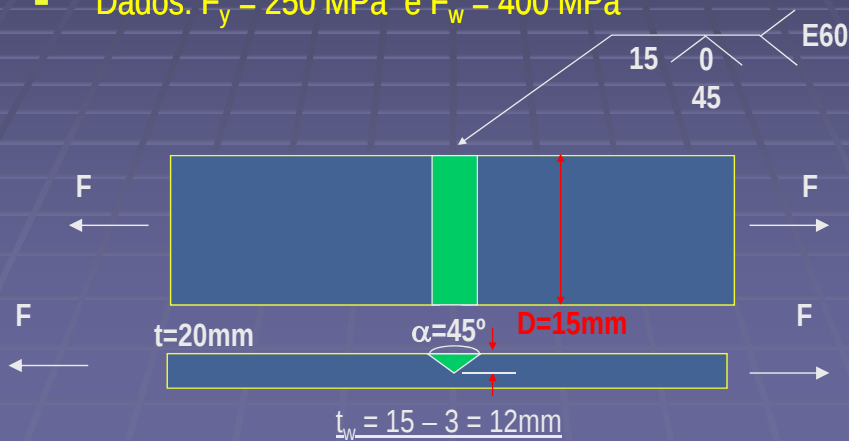
$$V_r = F / 2 = 0,15 \cdot 1140,1 = 171,92 \text{ kN} \rightarrow \quad \mathbf{F = 343,84 \text{ kN}}$$

4

14. Exemplos

2. Determinar o valor mínimo para a força F sabendo-se que foi utilizada solda de entalhe com penetração parcial

- Dados: $F_y = 250 \text{ MPa}$ e $F_w = 400 \text{ MPa}$



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14. Exemplos

$$V_r = \phi \cdot A_n \cdot F_u$$

$$A_n = t_w \cdot l_w = 12 \cdot 300 = 3600 \text{ mm}^2$$

$$V_r = 0,9 \cdot 3600 \cdot 0,400 = 1296 \text{ kN}$$

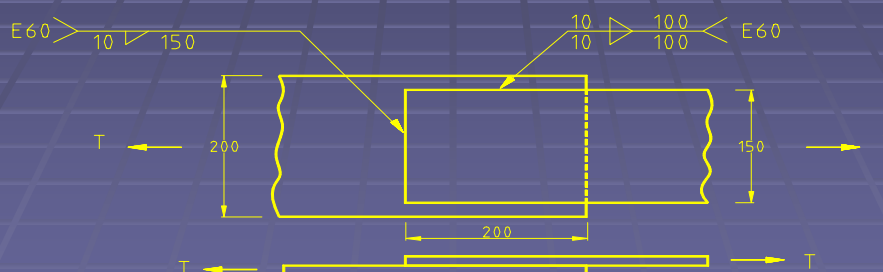
$$F = 1296 \text{ kN}$$

6

14. Exemplos

3. Determinar a resistência da ligação abaixo

Dados: $F_y = 250 \text{ MPa}$ e $F_w = 415 \text{ MPa}$



10  tw

$$Q_{\text{Lu}} \Delta w_{\text{Lu}} \cdot 0,67 = 0,67 \times 0,67 \times 0,415 \times 2474,5 = 460,98 \rightarrow \text{controle}$$

e60) $3 \nabla 4$

A diagram of a horizontal beam of length 10m. A triangular load is applied downwards, starting at 0 kN/m at the left end and increasing linearly to 10 kN/m at the right end. A point load $F = 500 \text{ kN}$ is applied downwards at the right end of the beam. The beam is supported by a pin support at the left end and a roller support at the right end. The triangular load is represented by a triangle with its base at the right end and its peak at the right end. The point load is represented by a downward arrow at the right end.

E60 5V 12

$$L_1 \times 22,6 = L_2 \times 57,4$$

$$L_1 = 2,54 L_2$$

$$\angle w = (\angle_1 + \angle_2) = 2,54 \angle_2 + \angle_2 = 3,54 \angle_2$$

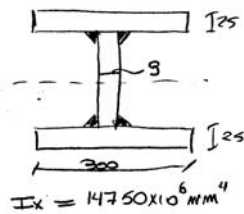
$$F = 50 \text{ kN}$$

$$\phi_{\Delta m} F_y \cdot 0,67 = 0,67 \times 0,9 \times 0,25 \times 5 \times 3,54 L_2 = 2,67 L_2$$

$$\phi_w A_w X_u \cdot 0,67 = 0,67 \times 0,67 \times 0,415 \times \frac{5 \times 9707}{t_w} \times 8,54 L_2 = 2,33 L_2$$

$$\begin{aligned} 2,33 L_2 &= 500 \quad \therefore L_2 = 214,18 \approx 215 \text{ mm} \approx 220 \text{ mm} \\ L_1 &= 2,54 k_e = 544,77 \approx 545 \text{ mm} \approx 550 \text{ mm} \end{aligned}$$

EXEMPLO 5



$$t = 5 \text{ mm}$$

$$E = 70 \text{ GPa}$$

$$\sigma_u = 495 \text{ MPa}$$

$$A_{g0} \text{ MR-250}$$

$$V = 800 \text{ kN}$$

$$q = \frac{V \cdot Q}{I} = \frac{800 \times 25 \times 300 \times (700 - 25/2)}{14750 \times 10^6}$$

$$q = 0,28 \text{ kN/mm}$$

$$V_r = 0,67 \times 0,9 \times 5.1 \cdot 0,25 = 0,75 \text{ kN/mm} \leftarrow \text{controle}$$

$$V_r = 0,67 \cdot 0,67 \times 5 \times 0,707 \cdot 1 \times 0,495 = 0,73 \text{ kN/mm}$$

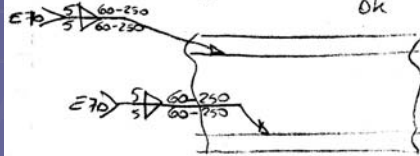
$$l_{min} \geq 4D = 4 \times 5 = 20 \text{ mm}$$

$$2 \times l = 60 \text{ mm}$$

$$\sim \text{espaçados de } 250 \text{ mm}$$

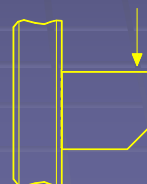
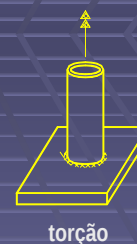
$$q = 0,75 \times \frac{60}{250} \times 2 = 0,36 > 0,28$$

Ok



15. Carregamento Excêntrico

- Se a relação força por comprimento de solda versus deslocamento for admitida linear $\rightarrow$ princípio de superposição é válido e o cálculo das ligações sujeitas a cisalhamento, torção e flexão combinados, pode ser feito isoladamente



15. Carregamento Excêntrico

- Taxa de força devido ao cortante V

$$q_v = \frac{V}{L}$$

- Taxa de força devido ao momento fletor M

$$q_m = \frac{M \cdot c}{I}$$

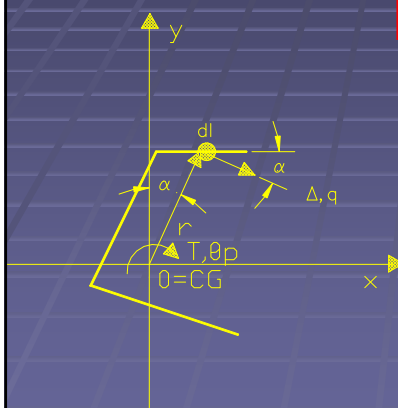
I é o momento de inércia do cordão de solda em relação ao eixo de flexão e c a distância deste eixo ao ponto da solda

- Taxa de força devido ao torsor T

$$q_t = \frac{T \cdot r}{I_p}$$

I_p é o momento polar de inércia da solda em relação ao seu centro geométrico, e r é a distância deste centro ao ponto da solda em consideração

15. Carregamento Excêntrico



$$\Delta = \theta_p \times r$$

$$q_t = k \times \Delta = k \times \theta_p \times r$$

fluxo / cordão de solda

$$T = \int_0^L q_t \cdot r \cdot dL = k \cdot \theta_p \int_0^L r^2 dL =$$

$$T = k \cdot \theta_p \cdot \bar{I}_p$$

15. Carregamento Excêntrico

$$q_t = \frac{T}{I_p} r$$

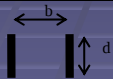
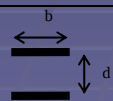
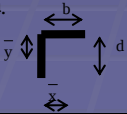
$$q_{tx} = \frac{T}{I_p} r \cdot \cos \alpha = \frac{T}{I_p} y$$

$$q_{ty} = \frac{T}{I_p} r \cdot \sin \alpha = \frac{T}{I_p} x$$

Roteiro:

1. $t_w \rightarrow$ seção geométrica
2. sistema de coordenadas no CG da solda
3. cisalhamento, flexão e torção na solda
4. carga / comprimento de solda $\rightarrow$ resistências
5. combinação vetorial $\rightarrow$ resistência final da solda

15. Carregamento Excêntrico

Seção b = largura e d = altura	$\bar{x}$ e $\bar{y}$	Módulo da seção $I_x / \bar{y}$	Momento Polar de Inércia em relação ao centro geométrico
1. 		$W = \frac{d^2}{6}$	$\bar{I}_p = \frac{d^3}{12}$
2. 		$W = \frac{d^2}{3}$	$\bar{I}_p = \frac{d(3b^2 + d^2)}{6}$
3. 		$W = b d$	$\bar{I}_p = \frac{b(3d^2 + b^2)}{6}$
4. 	$\bar{y} = \frac{d^2}{2(b+d)}$ $\bar{x} = \frac{b^2}{2(b+d)}$	$W = \frac{4bd + d^2}{6}$	$\bar{I}_p = \frac{(b+d)^2 - 6b^2d^2}{12(b+d)}$

15. Carregamento Excêntrico

5.		$\bar{x} = \frac{b^2}{2b+d}$	$W = bd + \frac{d^2}{6}$	$\bar{I}_p = \frac{8b^3 + 6bd^2 + d^3}{12} - \frac{b^4}{2b+d}$
6.		$\bar{y} = \frac{d^2}{b+2d}$	$W = \frac{2bd + d^2}{3}$	$\bar{I}_p = \frac{b^3 + 6b^2d + 8d^3}{12} - \frac{d^4}{2d+b}$
7.			$W = bd + \frac{d^2}{3}$	$\bar{I}_p = \frac{(b+d)^3}{6}$
8.		$\bar{y} = \frac{d^2}{b+2d}$	$W = \frac{2bd + d^2}{3}$	$\bar{I}_p = \frac{b^3 + 8d^3}{12} - \frac{d^4}{b+2d}$
9.			$W = bd + \frac{d^2}{3}$	$\bar{I}_p = \frac{b^3 + 3bd^2 + d^3}{6}$
10.			$W = \pi r^2$	$\bar{I}_p = 2\pi r^3$

15. Carregamento Excêntrico

- Processo de dimensionamento alternativo

$$\left(\frac{P}{P_0}\right)^2 + \left(\frac{M}{M_0}\right)^2 + \left(\frac{T}{T_0}\right)^2 + 2\left(\frac{T}{T_0}\right)\left(\frac{P}{P_0}\right)^2 \cos \varphi = 1$$

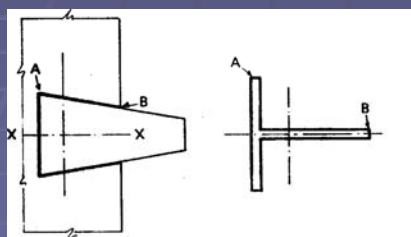


Figure 8.13 Examples of weld groups that do not conform to a conventional interaction relationship. A – critical points for moment about XX; B – critical points for torsion

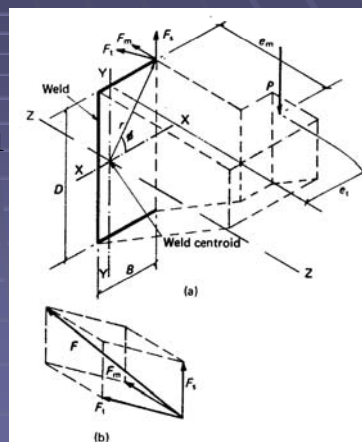


Figure 8.10 Weld group under combined shear, torsion and moment. (a) Overall analysis; (b) vector summation

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15. Carregamento Excêntrico

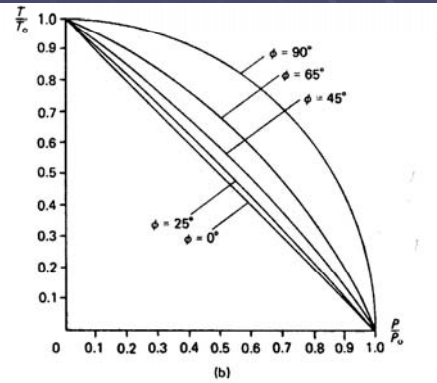
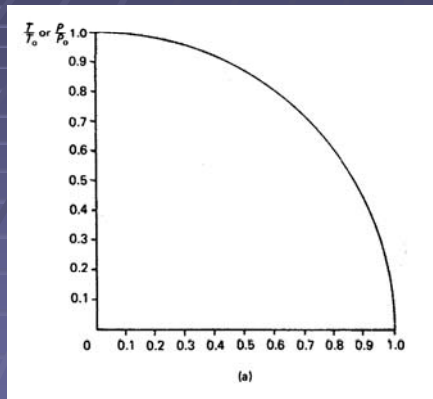


Figure 8.14 Interaction diagrams for weld groups. (a) P/P_0 or $T/T_0 = 0$; (b) $M/M_0 = 0$

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15. Carregamento Excêntrico

- $\phi = 90^\circ$ e $\phi = 65^\circ$

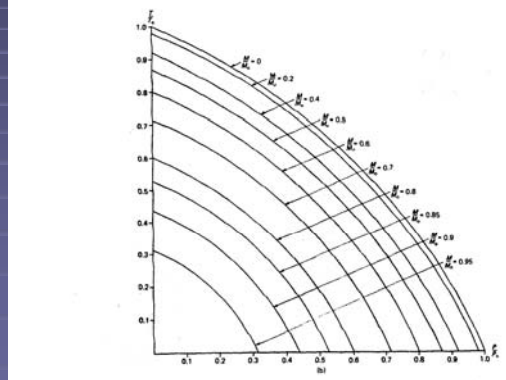
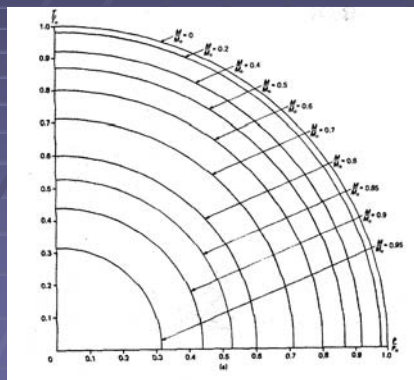


Figure 8.15 Interaction diagrams for weld groups. (a) $\phi = 90^\circ$; (b) $\phi = 65^\circ$

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15. Carregamento Excêntrico

- $\phi = 45^\circ$ e $\phi = 25^\circ$ e 0°

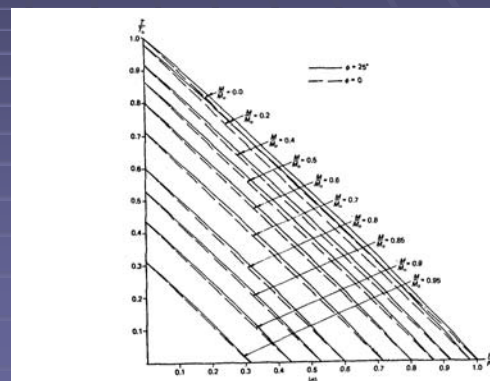
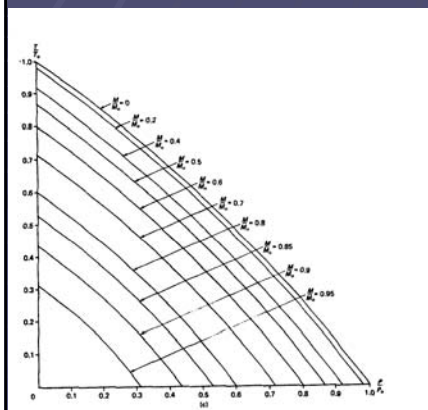


Figure 8.15 (cont.) (c) $\psi = 45^\circ$; (d) $\psi = 25^\circ$ and 0°

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15. Carregamento Excêntrico

- Cargas no Plano

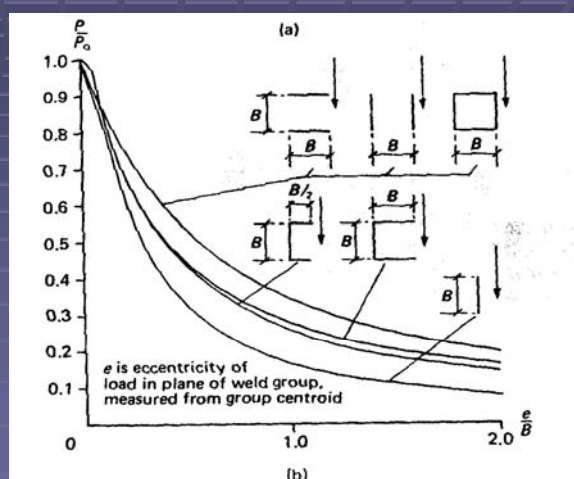
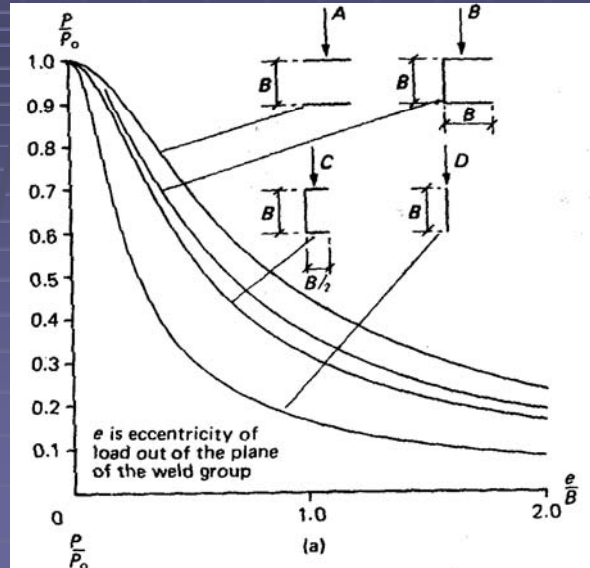


Figure 8.16 Interaction diagrams for weld groups. (a) Shear/moment diagrams, with no torsion; (b) shear/torsion diagrams, with no moment

15. Carregamento Excêntrico

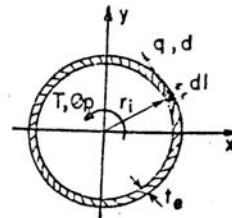
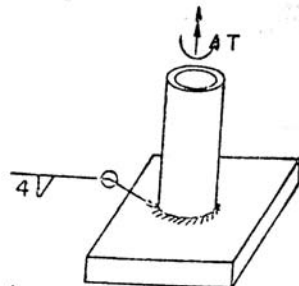
- Cargas fora do Plano



16. Exemplos com Cargas Excêntricas

EXEMPLO 6

DETERMINE O TORSOR DE PROSETO MÁXIMO DA LIGAÇÃO ABAIXO
 E60XX $X_u = 415 \text{ MPa}$ $F_y = 250 \text{ MPa}$, $F_u = 400 \text{ MPa}$
 $R_{w10} = 240 \text{ mm}$



$$t_w = 0,707 D = 0,707 \times 4 = 2,83 \text{ mm}$$

$$q_t = \frac{T r_i}{I_p}$$

$$I_p = 2\pi r_i^3$$

$$q_t = \frac{T d}{I_p} r_i = \frac{T \cdot 240}{8,69 \times 10^7}$$

$$= 2, \pi \cdot 240^3 = 8,69 \times 10^7 \text{ N mm}^3$$

$$= 2,76 \times 10^{-6} T_d$$

16. Exemplos com Cargas Excêntricas

$$I_p = \bar{I}_x + \bar{I}_y, \quad F_y = \frac{4 \times 120^3}{12} + 2 \times 150 \times 30^2 + 200 \times 45^2 + \left(\frac{200}{12}\right) = 1,24 \times 10^6$$

$$I_x = \frac{200^3}{12} + \left(2 \times \frac{150^3}{12}\right) + 2 \times 150 \times 100^2 = 3,67 \times 10^6 \text{ mm}^4, \quad I_p = 4,91 \times 10^6 \text{ mm}^4$$

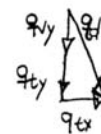
- Esforços

$$q_{vy} = \frac{75}{500} = 0,15 \text{ kN/mm} (\downarrow), \quad T = 75 \left(\frac{e}{L - \bar{x}} \right) = 22,9 \times 10^3 \text{ kN/mm}$$

$$q_{tx} = \frac{T_y}{I_p} = \frac{22,9 \times 10^3 \times 100}{4,91 \times 10^6} = 0,466 \text{ kN/mm} (\rightarrow)$$

$$q_{ty} = \frac{T_x}{I_p} = \frac{22,9 \times 10^3 \times 105}{4,91 \times 10^6} = 0,490 \text{ kN/mm} (\uparrow)$$

$$q_d = \sqrt{(0,466 + 0)^2 + (0,490 + 0,15)^2} = 0,791 \text{ kN/mm}$$



16. Exemplos com Cargas Excêntricas

- Resistência

$$A_w = 0,707 D \cdot t = 0,707 D$$

$$A_m = \frac{D \cdot t}{2} = \frac{D}{2}$$

$$0,67 \phi_w A_w X_u = 0,67 \cdot 0,67 \cdot 0,707 D \cdot 0,485 = 0,154 D \text{ kN/mm}$$

$$0,67 \phi A_m F_y = 0,67 \cdot 0,9 D \cdot 0,25 = 0,151 D \text{ kN/mm} \rightarrow \text{controla}$$

$$q_r = 0,151 D \text{ kN/mm}$$

$$q_r \geq q_d$$

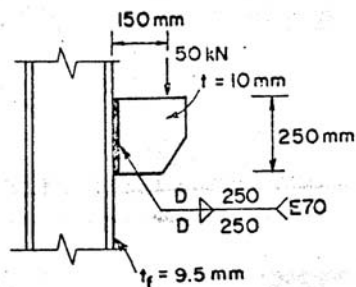
$$0,151 D \geq 0,791 \quad \therefore D \geq 5,25 \text{ mm}$$

$$t = 12 \text{ mm} \quad D_{\min} = 5 \text{ mm}$$

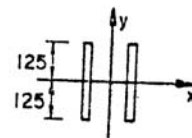
$$\text{use } D = 6 \text{ mm}$$

16. Exemplos com Cargas Excêntricas

EXEMPLO 8: Dimensione a ligação da figura abaixo



AÇO MR-25C



$$L = 2 \times 250 = 500 \text{ mm}$$

$$\overline{I_x} = \frac{2 \times 250^3}{12} = 260 \times 10^4 \text{ mm}^4$$

16. Exemplos com Cargas Excêntricas

- Esforços

$$q_{vy} = \frac{P}{L} = \frac{50}{500} = 0,1 \text{ kN/mm}$$

$$q_{mz} = \frac{M_c}{\overline{I_x}} = \frac{(50 \times 150) \times 125}{260 \times 10^4} = 360 \text{ N/mm} = 0,360 \text{ kN/mm}$$



$$q_d = \sqrt{(0,1)^2 + (0,36)^2} = 0,374 \text{ kN/mm}$$

Do exemplo 7 $q_r = 0,151 D$

$$0,151 D \geq 0,374 \quad \therefore D \geq 2,48 \text{ mm}$$

$t = 10 \text{ mm}$ (chapa + espinaço) $\rightarrow D_{\min} = 5 \text{ mm}$

$$\text{Use } D = 5 \text{ mm}$$